Appendix M

Space Thermal Analysis through Reduced Finite Element Modelling

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Abstract

The finite element method (FEM) is widely used in mechanical engineering, especially for space structure design. However, FEM is not yet often used for thermal engineering of space structures where the lumped parameter method (LPM) is still dominant.

LPM offers more accurate surfaces and fewer nodes to generate the radiative links while the FEM has automatic meshing tools and generation of conductive links. Coupled thermo-structural analyses are made straightforward if the same mesh can be used.

The proposed method brings together FEM and LPM by taking advantages on both sides. The structural FE mesh is reduced and the concept of super-node introduced. The reduction provides accurate conductive links and reduces the number of faces to compute the radiative links with Monte Carlo raytracing. The reduced model can integrate user logic in the exact same way a LPM model would do. Once the reduced model is solved using standard techniques, reduction matrices are exploited again to derive the detailed mesh temperatures for thermo-mechanical analyses.

To further reduce the computation time, quasi-Monte Carlo ray-tracing acceleration techniques were presented in the previous editions of the workshop, providing between 50% and one order of magnitude reduction of the number of rays required for a given accuracy. Combined with this acceleration technique, quadric surface fitting of selected regions in the FE mesh is performed to alleviate the FE mesh surface accuracy issue.

This presentation will summarise the research project developments carried out for the last four years. The end-to-end procedure will be detailed with actual space structures.





Reconciliation through a global approach

Radiative links computation

- Reduce # of rays: quasi-Monte Carlo method (isocell, Halton)
- Reduce # of facets: super-face concept (mesh clustering)

Surface accuracy for ray-tracing

Quadrics fitting

Conductive links, thermo-mech. analysis and user-defined compts.

- Reduce detailed FE mesh (keep conductive info. of the detailed geometry)
- Able to recover detailed T° from reduced
- Transform reduced FE model to LP model to enable user-defined comp.

Outline Ray-tracing enhancement FEM clustering & conductive reduction Super-face ray-tracing Integrating the developments Conclusions & perspectives



























Not picking a representative node of the cluster but creating new nodes

A super-node = weighted (area, volume) average each node cluster

$$T_{SN} = AT$$

$$T_{SN_i} = \sum_{j=1}^{N} A_{ij} T_j$$
 $\sum_{j=1}^{N} A_{ij} = 1$

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Reduction assumes uniform load

$$\begin{cases} K_{L}T = Q \\ T_{SN} = AT \end{cases} \iff \begin{bmatrix} K & A^{T} \\ A & 0 \end{bmatrix} \begin{bmatrix} T \\ 0 \end{bmatrix} = M \begin{bmatrix} T \\ 0 \end{bmatrix} = \begin{bmatrix} Q \\ T_{SN} \end{bmatrix}$$
$$\begin{cases} T \\ 0 \end{bmatrix} = M^{-1} \begin{bmatrix} Q \\ T_{SN} \end{bmatrix} = \begin{bmatrix} X & Y^{T} \\ Y & Z \end{bmatrix} \begin{bmatrix} Q \\ T_{SN} \end{bmatrix}$$
$$YA^{T} = I = AY^{T}$$
$$0 = YQ + ZT_{SN}$$

If the load is uniform over each super-node ($Q = A^T Q_{SN}$): $YQ = Q_{SN}$

$$-ZT_{SN} = Q_{SN}$$
$$K_{SN} = -Z$$

And the detailed T° can be recovered:

$$\mathbf{T} = \mathbf{X}\mathbf{Q} + \mathbf{Y}^{\mathrm{T}}\mathbf{T}_{\mathrm{SN}}$$











Number of reduced nodes





Save the attachment to disk or (double) click on the picture to run the movie.



Selective quadric fitting

Automatic quadric mesh fitting of user selected regions (e.g. optics)

$$f(\mathbf{x}) = \mathbf{C}^{\mathrm{T}}\mathbf{F} \qquad \mathbf{F}(\mathbf{x}) = [1, x, y, z, xy, xz, yz, x^{2}, y^{2}, z^{2}]^{\mathrm{T}}$$
$$\mathbf{C} = [c_{0}, \dots, c_{9}]^{\mathrm{T}}$$

$$error \approx \sum_{S_i \in R} \int_{S_i} \frac{f(\mathbf{x})^2}{|\nabla f(\mathbf{x})|^2} d\sigma \approx \frac{\mathbf{C}_0^{\mathrm{T}} M \mathbf{C}_0^{\mathrm{T}}}{\mathbf{C}_0^{\mathrm{T}} N \mathbf{C}_0^{\mathrm{T}}}$$

With

$$\mathbf{M} = \frac{1}{n} \sum_{\substack{i=1, \\ \mathbf{x}_i \in R}}^{n} \mathbf{F}(\mathbf{x}_i) \mathbf{F}(\mathbf{x}_i)^{\mathrm{T}} \qquad \mathbf{N} = \frac{1}{n} \sum_{\substack{i=1, \\ \mathbf{x}_i \in R}}^{n} \nabla \mathbf{F}(\mathbf{x}_i) \nabla \mathbf{F}(\mathbf{x}_i)^{\mathrm{T}}$$

C is the eigen vector associated with minimum eigen value of

 $M - \lambda N$













<section-header>CONCLUSIONS & PERSPECTIVES Integrated approach for detailed analysis of complex structures . Mo GMM needed, no T° mapping → time and costs saving! . Takes advantages of both lumped parameter and finite element methods: . More accurate conductive links . Accurate shape recognition used for ray-tracing . Reduce the gap between thermal and structural analyses . Despectives . Iterative process with automatic refinement in high ΔT regions . Quadric fitting → opto-thermo-structural analyses

Thank you for your attention...

Any question?

